## SSMC Survival Guide

"The answers to all the crazy Pneumatic and Fluid Power Questions"

## Fasf Delivery

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| Metric to English |  |  | English to Metric |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Multiply $\nabla$ | $\begin{gathered} \text { By } \\ \hline \end{gathered}$ | To Obtain $\nabla$ | Multiply $\nabla$ | $\begin{gathered} B y \\ \nabla \end{gathered}$ | To Obtain $\nabla$ |
|  |  | $\begin{array}{cl} \ldots \ldots \ldots & \text { mil } \\ \ldots \ldots & \text { in } \\ \cdots \cdots & \text { in } \\ \cdots \cdots & \mathrm{ft} \end{array}$ | Length: <br> mil $\qquad$ <br> in $\qquad$ <br> in $\qquad$ <br> ft $\qquad$ | $\begin{aligned} & 25.4 \ldots . . . . \\ & 25.4 . \ldots \\ & 2.54 \ldots . \\ & 0.3048 \end{aligned}$ | $\begin{aligned} & \mu \mathrm{m} \\ & \mathrm{~mm} \\ & \mathrm{~cm} \\ & \mathrm{~m} \end{aligned}$ |
| Area: |  | $\begin{array}{cc} \ldots \ldots & \mathrm{in}^{2} \\ \ldots \ldots . & \mathrm{in}^{2} \\ \ldots \ldots & \mathrm{ft}^{2} \end{array}$ | Area: <br> in $^{2}$ $\qquad$ <br> $\mathrm{in}^{2}$ $\qquad$ <br> $\mathrm{ft}^{2}$ $\qquad$ | $\begin{aligned} & 645.16 \\ & 6.4516 \\ & 0.0929 . \end{aligned}$ | $\begin{aligned} & \mathrm{mm}^{2} \\ & \mathrm{~cm}^{2} \\ & \mathrm{~m}^{2} \end{aligned}$ |
| Volume: |  |  | Volume: in ${ }^{3}$ $\mathrm{in}^{3}$ $\mathrm{ft}^{3}$ $\mathrm{ft}^{3}$ gal (US) | $\begin{aligned} & 16387 \ldots \\ & 16.387 \\ & 0.0283 \\ & 28.329 \\ & 3.785 \ldots \end{aligned}$ | $\begin{aligned} & \ldots . . \mathrm{mm}^{3} \\ & \ldots . . \mathrm{cm}^{3}(\mathrm{cc}) \\ & \ldots . \mathrm{m}^{3} \\ & \ldots . . . . \\ & \ldots \end{aligned}$ |
| Weight: <br> g | 0.0353 2.2046 $\ldots \ldots$ | $\cdots$ | Weight: oz <br> lb | $\begin{aligned} & 28.349 \\ & 0.4536 \end{aligned}$ | $\ldots \mathrm{kg}$ |
| Force: <br> gf $\qquad$ <br> kgf <br> N $\qquad$ | $\begin{aligned} & 2.205 \times 10^{-3} \\ & 2.2046 \ldots \ldots . . . . . . \\ & 0.2248 \end{aligned}$ | $=\cdots . . . . \mathrm{lbf}$ | Force: <br> lbf $\qquad$ <br> lbf $\qquad$ <br> lbf $\qquad$ | $\begin{aligned} & 453.6 \\ & 0.4536 \\ & 4.4482 \end{aligned}$ |  |
| Torque: <br> $\mathrm{N} \cdot \mathrm{m}$ | 0.7375 $7.223 \ldots \ldots .$. | (.......ftlb | Torque: $\mathrm{ft} \cdot \mathrm{lb}$ $\mathrm{ft} \cdot \mathrm{lb}$ | $\begin{aligned} & 1.3559 \\ & 0.1383 \end{aligned}$ | $\begin{aligned} & \mathrm{N} \cdot \mathrm{~m} \\ & \mathrm{~kg} \cdot \mathrm{~m} \end{aligned}$ |
|  |  |  | Pressure: <br> in $\left(\mathrm{H}_{2} \mathrm{O}\right)$ <br> in $(\mathrm{Hg})$ <br> psi <br> psi <br> psi <br> psi | $\begin{aligned} & 0.00254 \\ & 0.03518 \\ & 6.8947 \ldots . . \\ & 0.06894 \\ & 0.0703 \ldots . \\ & 0.00689 \end{aligned}$ |  |
| Energy: <br> $\mathrm{N} \cdot \mathrm{m}$ <br> J <br> MJ $\qquad$ |  | $\begin{gathered} \ldots . . . \mathrm{ft} \cdot \mathrm{lb} \\ \ldots \ldots . . \\ \ldots \mathrm{ft} \cdot \mathrm{lb} \\ \ldots . . . \\ \mathrm{kWh} \end{gathered}$ | Energy: <br> $\mathrm{ft} \cdot \mathrm{lb}$. <br> $\mathrm{ft} \cdot \mathrm{b}$ $\qquad$ <br> kWh $\qquad$ | $\begin{aligned} & 1.356 \\ & 1.356 \\ & 3.6 . . . . . \end{aligned}$ |  |
| Power: <br>  |  |  | Power: <br> $\mathrm{ft} \cdot \mathrm{lb} / \mathrm{s}$. hp. | $\begin{aligned} & 1.356 . \\ & 0.7457 \end{aligned}$ | W <br> KW |
| Flow Rate: <br> e/min ANR ........... 0.035 .......................... SCFM |  |  | Flow Rate: SCFM | 28.3.... | e/min ANR |
| Flow Coefficient: $\mathrm{mm}^{2}$....................... 0.0556 $\qquad$ |  |  | Flow Coeff Cv <br> Temperatu | t: <br> 18 ${ }^{\circ} \mathrm{C}=5 / 9$ | $. \mathrm{mm}^{2}$ |
| SSMC |  |  |  |  |  |

Fractonal Becmal Mimactereonversion ehar

$$
1 \mathrm{~mm}=0.03937 \prime \quad 0.01^{\prime \prime}=0.254 \mathrm{~mm} \quad 1 "=25.4 \mathrm{~mm}
$$

| Inch | Decimal | mm | Inch | Decimal | mm | Inch | Decimal | mm |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1/64 | 0.016 | 0.397 | 11/32 | 0.344 | 8.731 | 43/64 | 0.672 | 17.066 |
| 1/32 | 0.031 | 0.794 | 23/64 | 0.359 | 9.128 | 11/16 | 0.688 | 17.463 |
| 3/64 | 0.047 | 1.191 | 3/8 | 0.375 | 9.525 | 45/64 | 0.703 | 17.859 |
| 1/16 | 0.063 | 1.588 | 25/64 | 0.391 | 9.922 | 23/32 | 0.719 | 18.256 |
| 5/64 | 0.078 | 1.984 | 13/32 | 0.406 | 10.319 | 47/64 | 0.734 | 18.653 |
| 3/32 | 0.094 | 2.381 | 27/64 | 0.422 | 10.716 | 3/4 | 0.75 | 19.05 |
| 7/64 | 0.109 | 2.778 | 7/16 | 0.438 | 11.113 | 49/64 | 0.766 | 19.447 |
| 1/8 | 0.125 | 3.175 | 29/64 | 0.453 | 11.509 | 25/32 | 0.781 | 19.844 |
| 9/64 | 0.141 | 3.572 | 15/32 | 0.469 | 11.906 | 51/64 | 0.797 | 20.241 |
| 5/32 | 0.156 | 3.969 | 31/64 | 0.484 | 12.303 | 13/16 | 0.813 | 20.638 |
| 11/64 | 0.172 | 4.366 | 1/2 | 0.5 | 12.7 | 53/64 | 0.828 | 21.034 |
| 3/16 | 0.188 | 4.763 | 33/64 | 0.516 | 13.097 | 27/32 | 0.844 | 21.431 |
| 13/64 | 0.203 | 5.159 | 17/32 | 0.531 | 13.494 | 55/64 | 0.859 | 21.828 |
| 7/32 | 0.219 | 5.556 | 35/64 | 0.547 | 13.891 | 7/8 | 0.875 | 22.225 |
| 15/64 | 0.234 | 5.953 | 9/16 | 0.563 | 14.288 | 57/64 | 0.891 | 22.622 |
| 1/4 | 0.25 | 6.35 | 37/64 | 0.578 | 14.684 | 29/32 | 0.906 | 23.019 |
| 17/64 | 0.266 | 6.747 | 19/32 | 0.594 | 15.081 | 59/64 | 0.922 | 23.416 |
| 9/32 | 0.281 | 7.144 | 39/64 | 0.609 | 15.478 | 15/16 | 0.938 | 23.813 |
| 19/64 | 0.297 | 7.541 | 5/8 | 0.625 | 15.875 | 61/64 | 0.953 | 24.209 |
| 5/16 | 0.313 | 7.938 | 41/64 | 0.641 | 16.272 | 31/32 | 0.969 | 24.606 |
| 21/64 | 0.328 | 8.334 | 21/32 | 0.656 | 16.669 | 63/34 | 0.984 | 25.003 |


| $\mathbf{m m}$ | Inch |
| :---: | :---: |
| 0.1 | 0.0039 |
| 0.2 | 0.0079 |
| 0.3 | 0.0118 |
| 0.4 | 0.0157 |
| 0.5 | 0.0197 |
| 0.6 | 0.0236 |
| 0.7 | 0.0276 |
| 0.8 | 0.0315 |
| 0.9 | 0.0354 |
| 1 | 0.0394 |
| 10 | 0.3543 |
| 11 | 0.4331 |
| 12 | 0.4724 |
| 13 | 0.5118 |
| 14 | 0.5512 |
| 15 | 0.5906 |
| 16 | 0.6299 |
| 4 | 0.1181 |
| 5 | 0.1575 |
| 6 | 0.1969 |
| 17 | 0.6693 |
| 18 | 0.7087 |
| 7 | 0.2362 |
| 8 | 0.3150 |
| 19 | 0.7480 |
| 20 | 0.7874 |
| 21 | 0.8268 |
| 22 | 0.8661 |
| 23 | 0.9055 |
| 24 | 0.9449 |
| 25 | 0.9843 |


| PSI | kgf/cm ${ }^{2}$ | MPa | kPa | bar |
| :---: | :---: | :---: | :---: | :---: |
| 4 | . 28 | . 03 | 28 | 0.28 |
| 10 | . 70 | . 07 | 69 | 0.69 |
| 12 | . 84 | . 08 | 83 | 0.83 |
| 15 | 1.1 | . 10 | 103 | 1.03 |
| 20 | 1.4 | . 14 | 138 | 1.38 |
| 22 | 1.5 | . 15 | 152 | 1.52 |
| 25 | 1.8 | . 17 | 172 | 1.72 |
| 29 | 2.0 | . 20 | 200 | 2.00 |
| 30 | 2.1 | . 21 | 207 | 2.07 |
| 35 | 2.5 | . 24 | 241 | 2.41 |
| 36 | 2.6 | . 25 | 248 | 2.48 |
| 41 | 2.9 | . 28 | 282 | 2.83 |
| 45 | 3.2 | . 31 | 310 | 3.10 |
| 49 | 3.4 | . 34 | 338 | 3.38 |
| 55 | 3.9 | . 38 | 379 | 3.79 |
| 59 | 4.1 | . 41 | 407 | 4.07 |
| 65 | 4.6 | . 45 | 448 | 4.48 |
| 70 | 4.9 | . 48 | 483 | 4.83 |
| 75 | 5.3 | . 52 | 517 | 5.17 |
| 80 | 5.6 | . 55 | 552 | 5.52 |
| 86 | 6.0 | . 59 | 593 | 5.93 |
| 90 | 6.3 | . 62 | 621 | 6.21 |
| 96 | 6.7 | . 66 | 662 | 6.62 |
| 100 | 7.0 | . 69 | 689 | 6.89 |
| 103 | 7.2 | . 71 | 710 | 7.10 |
| 104 | 7.3 | . 72 | 717 | 7.17 |
| 110 | 7.7 | . 76 | 758 | 7.58 |
| 115 | 8.1 | . 79 | 793 | 7.93 |
| 120 | 8.4 | . 83 | 827 | 8.27 |
| 125 | 8.8 | . 86 | 861 | 8.62 |
| 131 | 9.2 | . 90 | 903 | 9.03 |
| 135 | 9.5 | . 93 | 931 | 9.31 |
| 141 | 9.9 | . 97 | 972 | 9.72 |
| 145 | 10.2 | 1.0 | 1000 | 10.00 |
| 159 | 11.2 | 1.1 | 1096 | 10.96 |
| 215 | 15.1 | 1.48 | 1482 | 14.82 |

Theoretical Force $=$ Area $x$ Pressure

| Bore | Piston Area (in²) | Operating Pressure (psi) |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 25 psi | 50 psi | 75 psi | 100 psi | 125 psi | 150 psi |
| 1/4" (6mm) | 0.05 | 1 lbf | 2 lbf | 4 lbf | 5 lbf | 6 lbf | 7 lbf |
| 8mm | 0.08 | 2 | 4 | 6 | 8 | 10 | 12 |
| 3/8" (10mm) | 0.11 | 3 | 6 | 8 | 11 | 14 | 17 |
| 5/8" (16mm) | 0.31 | 8 | 15 | 23 | 31 | 38 | 46 |
| 3/4" (20mm) | 0.44 | 11 | 22 | 33 | 44 | 55 | 66 |
| 1" (25mm) | 0.79 | 20 | 39 | 59 | 79 | 98 | 118 |
| 1 1/8" | 0.99 | 25 | 50 | 75 | 99 | 124 | 149 |
| 30 mm | 1.10 | 27 | 55 | 82 | 110 | 137 | 164 |
| $11 / 4$ " $(32 \mathrm{~mm})$ | 1.23 | 31 | 61 | 92 | 123 | 153 | 184 |
| $11 / 2{ }^{1 / 2}(40 \mathrm{~mm})$ | 1.77 | 44 | 88 | 133 | 177 | 221 | 265 |
| $13 / 4$ " | 2.41 | 60 | 120 | 180 | 241 | 301 | 361 |
| 2" ( 50 mm ) | 3.14 | 79 | 157 | 236 | 314 | 393 | 471 |
| $21 / 2{ }^{1 / 2}(63 \mathrm{~mm})$ | 4.91 | 123 | 245 | 368 | 491 | 614 | 736 |
| 3114 (80mm) | 8.30 | 207 | 415 | 622 | 830 | 1037 | 1244 |
| 4" (100mm) | 12.57 | 314 | 628 | 942 | 1257 | 1571 | 1885 |
| $41 / 2$ " | 15.90 | 398 | 795 | 1193 | 1590 | 1988 | 2386 |
| 5" (125mm) | 19.63 | 491 | 982 | 1473 | 1963 | 2454 | 2945 |
| 140 mm | 23.86 | 597 | 1193 | 1790 | 2386 | 2983 | 3579 |
| 6" | 28.27 | 707 | 1414 | 2121 | 2827 | 3534 | 4241 |
| 160mm | 31.16 | 779 | 1558 | 2337 | 3116 | 3896 | 4675 |
| 7"' (180mm) | 38.48 | 962 | 1924 | 2886 | 3848 | 4811 | 5773 |
| 8" (200mm) | 50.27 | 1257 | 2513 | 3770 | 5027 | 6283 | 7540 |
| 10" (250mm) | 78.54 | 1963 | 3927 | 5890 | 7854 | 9817 | 11781 |
| 12" | 113.10 | 2827 | 5655 | 8482 | 11310 | 14137 | 16965 |

Note: Do not forget to apply safety factor of 0.7 for horizontal and 0.5 for vertical cylinder orientation.

## 

Cylinder Bore (inches)

| $\mathrm{ln} / \mathrm{sec}$ | 1/2 | 3/4 | 1 | 11/2 | 2 | $21 / 2$ | $31 / 4$ | 4 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | . 0014 | . 0032 | . 0058 | . 013 | . 023 | . 036 | . 061 | . 092 |
|  | . 041 | . 091 | . 16 | . 37 | . 65 | 1.0 | 1.73 | 2.6 |
| 2 | . 0029 | . 0065 | . 012 | . 026 | . 046 | . 072 | 12 | . 18 |
|  | . 081 | . 18 | . 33 | . 74 | 1.3 | 2.0 | 3.5 | 5.2 |
| 3 | . 0043 | . 0097 | . 17 | . 039 | 11 | . 069 | 18 | . 276 |
|  | . 13 | . 28 | . 5 | 1.1 | 3.0 | 2.0 | 5.2 | 7.8 |
| 4 | . 0058 | . 013 | . 023 | . 052 | . 092 | . 14 | . 24 | . 37 |
|  | 16 | . 37 | . 65 | 1.5 | 2.6 | 4.0 | 6.9 | 10.4 |
| 5 | . 0069 | . 015 | . 028 | . 065 | . 11 | . 18 | . 3 | . 46 |
|  | . 21 | . 46 | . 83 | 1.9 | 3.3 | 5.0 | 8.6 | 13.0 |
| 6 | . 0087 | . 020 | . 035 | . 078 | . 14 | . 22 | . 37 | . 55 |
|  | . 25 | . 56 | 1.0 | 2.2 | 4.0 | 6.1 | 10.4 | 15.6 |
| 7 | . 010 | . 023 | . 04 | . 091 | . 16 | . 25 | . 43 | . 64 |
|  | . 28 | . 44 | 1.13 | 2.6 | 4.5 | 7.1 | 12.1 | 18.2 |
| 8 | . 011 | . 025 | . 045 | . 10 | . 18 | . 29 | 49 | . 74 |
|  | . 33 | . 73 | 1.3 | 3.0 | 5.2 | 8.1 | 13.8 | 20.8 |
| 9 | . 013 | . 030 | . 053 | . 12 | . 21 | . 32 | . 55 | . 83 |
|  | . 36 | . 82 | 1.45 | 3.3 | 5.8 | 9.1 | 15.6 | 23.4 |
| 10 | . 014 | . 032 | . 058 | . 13 | . 23 | . 36 | . 61 | . 92 |
|  | . 36 | . 91 | 1.63 | 3.7 | 6.5 | 10.1 | 17.3 | 26.0 |
| 11 | . 016 | . 035 | . 063 | . 14 | . 25 | . 40 | . 67 | 1 |
|  | . 44 | 1.0 | 1.78 | 4.1 | 7.1 | 11.1 | 19.0 | 28.6 |
| 12 | . 018 | . 039 | . 07 | . 16 | . 28 | .43 | . 73 | 1.1 |
|  | . 49 | 1.1 | 1.8 | 4.4 | 7.8 | 12.1 | 20.8 | 31.2 |
| 13 | . 019 | . 042 | . 075 | . 17 | . 30 | . 47 | . 79 | 1.2 |
|  | . 53 | 1.18 | 2.10 | 4.8 | 8.4 | 13.1 | 22.5 | 33.8 |
| 14 | . 02 | . 045 | . 08 | . 18 | . 32 | . 50 | . 85 | 1.3 |
|  | . 57 | 1.28 | 2.28 | 5.2 | 9.1 | 14.1 | 24.2 | 36.4 |
| 15 | . 021 | . 048 | . 085 | . 19 | . 34 | . 54 | . 91 | 1.4 |
|  | 61 | 1.36 | 2.43 | 5.6 | 9.7 | 15.1 | 25.9 | 39.0 |
| 16 | . 023 | . 052 | . 093 | . 20 | . 37 | . 58 | . 98 | 1.5 |
|  | . 65 | 1.46 | 2.6 | 5.9 | 10.4 | 16.2 | 27.7 | 41.6 |
| 17 | . 024 | . 055 | . 096 | . 22 | . 39 | . 61 | 1.0 | 1.6 |
|  | . 69 | 1.55 | 2.75 | 6.3 | 11.0 | 17.2 | 29.4 | 44.2 |
| 18 | . 026 | . 058 | . 103 | . 230 | . 41 | . 65 | 1.1 | 1.7 |
|  | . 73 | 1.65 | 2.93 | 6.6 | 11.7 | 18.2 | 31.1 | 46.8 |
| 19 | . 028 | . 062 | . 11 | . 25 | . 44 | . 68 | 1.2 | 1.75 |
|  | . 77 | 1.73 | 3.08 | 7.0 | 12.3 | 19.2 | 32.8 | 49.4 |
| 20 | . 029 | . 065 | . 12 | . 26 | . 46 | . 72 | 1.25 | 1.8 |
|  | . 81 | 1.83 | 3.25 | 7.4 | 13.0 | 20.2 | 34.6 | 52.0 |
| 22 | . 032 | . 072 | . 13 | . 29 | . 51 | .79 | 1.3 | 2.0 |
|  | . 89 | 2.01 | 3.58 | 8.1 | 14.3 | 22.2 | 38.1 | 57.2 |
| 24 | . 034 | . 077 | . 14 | . 31 | . 55 | . 86 | 1.5 | 2.2 |
|  | 98 | 2.19 | 3.90 | 8.9 | 15.6 | 24.2 | 41.5 | 62.4 |
| 26 | . 037 | . 084 | . 15 | . 34 | . 60 | . 94 | 1.6 | 2.4 |
|  | 1.06 | 2.38 | 4.23 | 9.6 | 16.9 | 26.3 | 45.0 | 67.6 |
| 28 | . 04 | . 09 | . 16 | . 36 | . 64 | 1.0 | 1.7 | 2.6 |
|  | 1.14 | 2.56 | 4.55 | 10.3 | 18.2 | 27.3 | 48.4 | 72.8 |
| 30 | . 069 | . 097 | . 17 | . 39 | . 69 | 1.1 | 1.8 | 2.8 |
|  | 1.22 | 2.74 | 4.88 | 11.1 | 19.5 | 30.3 | 51.9 | 78.0 |

Cv Required at the cylinder Top / SCFM Lower: Cv based on 70 psi inlet and 10 psi pressure drop.
Note: This chart does not take into account the flow restrictions through the valve and tubing, etc..

Area ( in $^{2}$ ) $=$ diameter ${ }^{2} \times 0.7854$ or $\pi \mathrm{r}^{2}$
Circumference $=\pi \mathrm{D}=2 \pi \mathrm{r}$
Pressure = Force / Area
Force $=$ Pressure - Area
Cylinder Volume (Head end) = Piston Area • Stroke
Cylinder Volume (Rod end) $=($ Piston Area - Rod Area) • Stroke
Compression Ratio $($ C.R. $)=(\mathrm{psig}+14.7) / 14.7$
Consumption (Standard ft ${ }^{3}$ ) $=\left(\right.$ Area in ${ }^{2} \times$ Stroke in $\times$ Compression Ratio) $/ 1728$
Air Demand (scfm) $=60 \times$ Area $\mathrm{in}^{2} \times$ Piston Speed in/s $\times$ C.R.) $/ 1728$
Peak Air Flow (Q) = Volume / Time •C.R.
Torque $=$ Force $\bullet$ Perpendicular distance from shaft
Water Weight $=$ Pounds $=$ US Gallons $\times 8.3453$
$\pi=3.14, \mathrm{D}=$ Diameter, $\mathrm{r}=$ Radius

## Vatrestiting

Use the formula below with the cylinder flow chart above and the Compression Ratio and Pressure Drop Factor chart below to calculate the required Cv for a valve.

$$
\mathrm{C}_{\mathbf{v}}=\frac{\text { Piston Area }\left(\mathrm{in}^{2}\right) \times \text { Stroke (in) } \times \text { Compression Ratio }}{\text { Pressure Drop Factor } \times \text { Stroke Time }(\mathrm{sec}) \times 29}
$$

| Inlet <br> Pressure | Compression <br> Ratio | Pressure Drop Factors for Various <br> Pressure Drops |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 2 psi | 5 psi | 10 psi | 15 psi | 20 psi |
| $\mathbf{1 0}$ |  | 6.5 |  |  |  |  |
| $\mathbf{2 0}$ |  | 7.8 | 11.8 |  |  |  |
| $\mathbf{3 0}$ |  | 8.9 | 13.6 | 18.0 |  |  |
| $\mathbf{4 0}$ |  | 9.9 | 15.3 | 20.5 | 23.6 |  |
| $\mathbf{5 0}$ | 4.4 | 10.8 | 16.7 | 22.6 | 26.4 | 29.0 |
| $\mathbf{6 0}$ | 5.1 | 11.7 | 18.1 | 24.6 | 29.0 | 3.0 |
| $\mathbf{7 0}$ | 5.8 | 12.5 | 19.3 | 26.5 | 31.3 | 34.8 |
| $\mathbf{8 0}$ | 6.4 | 13.2 | 20.5 | 28.2 | 33.5 | 3.4 |
| $\mathbf{9 0}$ | 7.1 | 13.9 | 21.6 | 29.8 | 3.5 | 3.9 |
| $\mathbf{1 0 0}$ | 7.8 | 14.5 | 22.7 | 31.3 | 37.4 | 42.1 |
| $\mathbf{1 1 0}$ | 8.5 | 15.2 | 23.7 | 32.8 | 39.3 | 4.3 |
| $\mathbf{1 2 0}$ | 9.2 | 15.8 | 24.7 | 34.2 | 41.0 | 4.4 |
| $\mathbf{1 3 0}$ | 9.8 | 16.4 | 25.6 | 35.5 | 42.7 | 48.4 |
| $\mathbf{1 4 0}$ | 10.5 | 16.9 | 26.5 | 36.8 | 44.3 | 50.3 |

Note: Pressure drop factor is based on the inlet pressure of the valve and the allowable pressure drop across the valve. For average conditions use a 70 psi inlet pressure and a 10 psi pressure drop.
Note: For more accurate valve sizing, particularly when temperature is a factor, or the operation is speed critical, use the following procedure.

## Something to remember when choosing which equation to use for valve selection

1. In many instances temperature is not a factor in system applications. In most industrial application, compressed air temperature is roughly the same as ambient. If this is the case, then the use of equation $\# 1$ is recommended. This equation has been widely accepted to get a Cv value.
2. If temperature is a factor in the application then equation \#2 is recommended. We have chosen to use the constant 22.48 in our equations, but those who choose to be more conservative may choose use 22.67 as the constant. Both tied to ambient temperature.
3. When sizing a valve by calculating the Cv value, determining the pressure drop across the valve (i.e. $\Delta \mathrm{P}$ ), is an important step. What has proven to be a good practice in calculating Cv is the following:
a. For general applications use 10 psi for the pressure drop.
b. When a more conservative approach is needed, use 5 psi for the pressure drop.
c. If cylinder and design factors are critical, then using a 2 psi drop will more conservatively size the valve.
4. Also remember that, for calculation purposes, whether P1 is given in PSIG or PSIA, P2 needs to be reflected in absolute or PSIA (i.e. P2a)
5. Lastly, we recognize that not all applications will have a supply pressure of a higher valve: thus it is suggested that if P 1 is 60 PSI or less, a 5 PSI pressure drop across the valve be used to calculate the Cv value.
(Eq. 1) Simplified equation when temperature is not a factor

$$
\mathrm{Cv}=\frac{1.024 \times \mathrm{Q}}{\sqrt{\Delta \mathrm{P} \times \mathrm{P} 2 \mathrm{a}}}
$$

Given: $\mathrm{Cv}=$ Flow coefficient
1.024 = Constant

Q = Peak Flow Rate in SCFM
$\Delta \mathrm{P} \quad=$ Pressure drop across the valve
(See information above)
P2a = Down-stream (valve's outlet ) pressure in PSIA
(Eq. 2) Equation used when temperature is a factor in system application
$C v=\binom{Q}{22.48} \sqrt{\Delta \mathrm{P} \times \mathrm{P} 2 \mathrm{a}}$
Given: Cv = Flow coefficient
$22.48=$ Constant (22.7 is often used, but 22.48 will be used on the PS exam)
TR $=$ Temperature in Rankin ( ${ }^{\circ} \mathrm{F}+460$ )
Q = Peak flow retain SCFM
$\Delta \mathrm{P} \quad=$ Pressure drop across the valve (See information above)
P2a = Down-stream (valve's outlet) pressure in PSIA

## Vactume

Use the theoretical lift force (Ft) table below to determine what size vacuum cup to use for an application. Practical lift force $(\mathrm{Fp})$ should be calculated with the following formula. Use the safety factors ( t ) from the table.

$$
F_{p}=F_{t x} \times 1 / t
$$

| PLANE OF CUP CONTACT | STATIC LOAD | DYNAMIC LOAD |
| :--- | :---: | :---: |
| Horizontal | $\mathrm{t}>4$ | $\mathrm{t}>4$ |
| Vertical | $\mathrm{t}>4$ | $\mathrm{t}>8$ |


| Ft (lbf) |  | Vacuum Pressure ( $\mathbf{I n H g}$ ) |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Cup ø (mm) | Area ( $\mathrm{mm}^{2}$ ) | 26" | 24" | 22" | 20" | 18" | 16" | 14" | 12" |
| 2 | . 031 | . 062 | . 057 | . 05 | . 049 | . 042 | . 037 | . 033 | . 029 |
| 4 | . 126 | . 245 | . 225 | . 207 | . 187 | . 170 | . 150 | . 132 | . 112 |
| 6 | . 283 | . 551 | . 509 | 465 | . 423 | . 381 | . 340 | . 298 | . 254 |
| 8 | . 503 | . 979 | 904 | . 829 | 754 | . 677 | . 602 | . 527 | . 452 |
| 10 | . 785 | 1.53 | 1.41 | 1.29 | 1.18 | 1.06 | . 941 | . 825 | . 705 |
| 13 | 1.33 | 2.58 | 2.38 | 2.18 | 1.98 | 1.79 | 1.59 | 1.39 | 1.19 |
| 16 | 2.01 | 3.90 | 3.62 | 3.31 | 3.02 | 2.71 | 2.40 | 2.12 | 1.81 |
| 20 | 3.14 | 6.13 | 5.64 | 5.16 | 4.70 | 4.23 | 3.77 | 3.31 | 2.82 |
| 25 | 4.91 | 9.57 | 8.82 | 8.09 | 7.36 | 6.61 | 5.89 | 5.14 | 4.41 |
| 32 | 8.04 | 15.7 | 14.5 | 13.3 | 12.1 | 10.8 | 9.63 | 8.44 | 7.23 |
| 40 | 12.6 | 24.5 | 22.5 | 20.6 | 18.8 | 16.9 | 15.1 | 13.2 | 11.3 |
| 50 | 19.6 | 38.1 | 35.3 | 32.4 | 29.3 | 26.5 | 23.6 | 20.6 | 17.7 |

Note: If several cups are used simply add up the forces for each cup

## Shing Yacubin fectors

Step 1 - Determine values for adsorption response time. $T_{1} \& T_{2}$

Pave $=$ Vacuum pressure required.
Given: $T_{2}=$ Adsorption response time to $95 \%$ of Pave. (Time Required by process)
Find: $T_{1}=$ Adsorption response time to $63 \%$ of Pave. $T_{1}=\frac{T_{2}}{3}$

Step 2 - Determine the total volume of your system by calculating component volumes and adding them together.

2a) Tube Volume $\mathrm{V}_{\mathrm{t}}$ ( $\mathrm{mm}^{3}$ )
$V_{t}=\frac{\pi}{4} \times d^{2} x \frac{L}{1000}$

Where:
$\mathrm{V}_{\mathrm{t}}=$ tube volume $\left(\mathrm{mm}^{3}\right)$
$\mathrm{d}=\mathrm{ID}$ of tube (mm)
$L=$ Length of tube ( $m$ )

## Step 2 - Determine the total volume of your system by calculating component volumes and adding them together. (Continued)

2b) Pad Volume $\mathrm{V}_{\mathrm{p}}$ (if significant): ( $\mathrm{mm}^{3}$ )
$V_{p}=\frac{\pi}{4} \times d^{2} \times L$

> Where: $\begin{aligned} & V_{p}=\text { pad volume }\left(\mathrm{mm}^{3}\right) \\ & d=I D \text { of pad }(\mathrm{mm}) \\ & L=\text { Depth of pad }(\mathrm{mm})\end{aligned}$

## For bellows pads

$$
V_{p b}=\frac{\pi}{4} \times A^{2} \times Y
$$

Where:
$\mathrm{V}_{\mathrm{bp}}=$ pad volume $\left(\mathrm{mm}^{3}\right)$
A = Dimension A from chart ID of Pad (mm)
$\mathrm{Y}=$ Dimension Y from chart Depth of Pad (mm)

| Model | A | B | D | H: M6 x 1 |  |  |  |  | H: $\mathrm{M} 8 \times 1$ |  |  |  |  | K | L | Y |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | C | E | F | G | I | C | E | F | G | I |  |  |  |
| ZPT20B | 20 | 22 | 23.5 | 3 | 28.5 | 25 | 54.5 | 8 | 3.5 | 33.5 | 15 | 49.5 | 12 | 12 | 25 | 10.5 |
| ZPT25B | 25 | 27 | 24 |  | 29 |  | 55 |  |  | 34 |  | 50 |  | 16 | 28 | 10.5 |
| ZPT32B | 32 | 34 | 29 |  | 34 |  | 60 |  |  | 39 |  | 55 |  | 19 | 37 | 14 |



2c) Buffer Volume $\mathrm{V}_{\mathrm{b}}$ (if present)
To approximate using $\mathrm{C}, \mathrm{G}, \& \mathrm{Y}$ in the standard equation for volume:
$\mathrm{V}_{\mathrm{p}}=\frac{\pi}{4} \times \mathrm{C}^{2} \times(\mathrm{G}-\mathrm{Y})$
2d) Filter Volume $\mathrm{V}_{\mathrm{f}}$ (if present) ( $\mathrm{mm}^{3}$ )
Consult Best Pneumatics (for example):

$$
\begin{aligned}
& \text { AMJ3000 }=30 \mathrm{cc}^{*} 1000=30,000 \mathrm{~mm}^{3} \\
& \text { AMJ4000/5000 }=85 \mathrm{cc}^{*} 1000=85,000 \mathrm{~mm}^{3}
\end{aligned}
$$

Or calculate approximate filter volume by dimensions from the catalog.
(Note that dimensions are not always given for the ID of the filter, so estimate can be used)
Where:

$$
V_{f}=\frac{\pi}{4} \times d^{2} x h
$$

$\mathrm{V}_{\mathrm{f}}=$ filter volume $\left(\mathrm{mm}^{3}\right)$
$\mathrm{d}=\mathrm{ID}$ of filter (mm)
$\mathrm{Y}=$ height of filter ( mm )

2e) Add component volumes together ( $\mathrm{mm}^{3}$ )
$\mathrm{V}_{\text {total }}=\mathrm{V}_{\mathrm{t}}+\mathrm{V}_{\mathrm{p}}+\mathrm{V}_{\mathrm{b}}+\mathrm{V}_{\mathrm{f}}+\mathrm{V}_{\text {misc }}$

2f) Convert form $\mathrm{mm}^{3}$ to Liters
$V_{\text {total }}\left(\mathrm{mm}^{3}\right) \times \frac{1(\text { Liter })}{1,000,000\left(\mathrm{~mm}^{3}\right)}=\mathrm{V}_{\text {total }}$ (Liters)

Step 3 - Determine the mean vacuum flow, $Q_{1}$ (liter/mm)

$$
Q_{1}=\frac{V_{\text {total }}}{T_{1}} \times \frac{60 \mathrm{sec}}{1 \mathrm{~min}}
$$

Where: Q1 = Average flow required ( $\mathrm{L} / \mathrm{min}$ )
$\mathrm{V}_{\text {total }}=$ Volume of to be evacuated (liters)
Step 4 - Determine Leakage, $Q_{\llcorner }$(Liter/min) and $Q_{\text {max }}$ (liter/min)

Connect pad to a test ejector and vacuum pressure gauge. Operate ejector at recommended supply pressure and place pad on work piece.

Note the vacuum pressure achieved and compare it to chart from the catalog for the ejector.

If the pressure gauge shows full vacuum pressure achieved, then there is no leakage.
Then use $Q_{\text {max }}=2 \times Q_{1}$
If the pressure gauge shows less than full vacuum pressure achieved, determine QL by finding pressure achieved on graph. Move to the right until intersecting diagonal line above the QL flow rate
Then use $Q_{\text {max }}=3 \times\left(Q_{1}+Q_{\llcorner }\right)$

Flow Characteristics


Vacuum flow rate scfm (L/min (ANR))

## Step 5 - Choose ejector.

Choose an ejector that meets the physical characteristics, optional features and $Q$ max flow rate that will perform adsorption in the given time. $\mathrm{T}_{2}$


ZM



ZH

ZZM


## Pre threst auick peferche

Tapered pipe threads seal at the points where the crests of the threads meet the roots of the mating threads. Standard pipe threads, NPT, PT, and BSPT require sealant to prevent the development of a spiral leak path. NPTF threads are designed to crush the points of the crests into the roots of the mating threads to achieve the same purpose, however, use of a lubricant or sealant to prevent galling of the threads is preferred where not functionally prohibited.

BSPT - British Standard Taper Pipe Threads
PT - Japanese Industrial Standard Taper Pipe Threads
\{R (PT) - Taper external threads\}
\{Rc (PT) - Taper internal threads\}
NPT - American National Standard Taper Pipe Threads
*All of the above are designed to be used with sealant to provide a pressure tight joint.
NPTF - American National Standard Dry seal Pipe Threads
*Designed to provide a pressure tight joint without the use of sealant.
PF - Japanese Industrial Standard Parallel Pipe Threads
*Straight threads use a gasket or O-ring to produce a pressure tight joint.
Basic Dimensions

| Port <br> Size | PT \& BSPT |  |  |  | NPT \& NPTF |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Threads <br> per inch | Pitch | Major <br> Dia. | Thread <br> form <br> angle | Threads <br> per inch | Pitch | Major <br> Dia. | Thread <br> form <br> angle |
|  | 28 | .03571 | .304 | $55^{\circ}$ | 27 | .030704 | .313 | $60^{\circ}$ |
| $\mathbf{1 / 8}$ | 28 | .03571 | .383 | $55^{\circ}$ | 27 | .030704 | .404 | $60^{\circ}$ |
| $\mathbf{1 / 4}$ | 19 | .05262 | .518 | $55^{\circ}$ | 18 | .05556 | .540 | $60^{\circ}$ |
| $\mathbf{3 / 8}$ | 19 | .05262 | .656 | $55^{\circ}$ | 18 | .05556 | .675 | $60^{\circ}$ |
| $\mathbf{1 / 2}$ | 14 | .07142 | .825 | $55^{\circ}$ | 14 | .07143 | .840 | $60^{\circ}$ |
| $\mathbf{3 / 4}$ | 14 | .07142 | 1.041 | $55^{\circ}$ | 14 | .07143 | 1.050 | $60^{\circ}$ |

Compatibility between the above male and female is outlined below. SMC Corporation, however, has the unique solution to all this complexity. The Uni-Fit will screw into all major thread variations.

|  |  | Female |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Parallel |  |  |  | Taper |  |  | American |  |
|  |  | BSP | Rp | PF | G | BSPT | Rc | PT | NPT | NPTF |
| $\frac{0}{\sum_{\sum}^{0}}$ | BSP | Y | Y | Y | Y | N | N | N | N | N |
|  | BSPT | Y | Y | Y | Y | Y | Y | Y | N | N |
|  | G | Y | Y | Y | Y | N | N | N | N | N |
|  | NPT | N | N | N | N | N | N | N | Y | N |
|  | NPTF | N | N | N | N | N | N | N | N | Y |
|  | PF | Y | Y | Y | Y | N | N | N | N | N |
|  | PT | Y | Y | Y | Y | Y | Y | Y | N | N |
|  | R | Y | Y | Y | Y | Y | Y | Y | N | N |
|  | UNI | Y | Y | Y | Y | Y | Y | Y | Y | Y |

Miniature threads, M5x0.8 and 10/32 UNF, will only mate as follows: $10 / 32$ male will fit into an M5 female, M5 male will NOT fit into a 10/32 female. Both of these threads use a gasket to produce a pressure tight fit.
"Standard ISO port call out"

| Port ID | Description of Function |
| :---: | :--- |
| $\mathbf{1}$ | Inlet - Supply Pressure $\{$ Port P\} |
| $\mathbf{2}$ | Output - Normally Open at rest (Unless specified in a 2 or 3 port <br> valve) $(1 \rightarrow-2)\{$ Port B $\}$ |
| $\mathbf{4}$ | Output - Normally Closed at rest $(4 \rightarrow 5)\{$ Port A $\}$ |
| $\mathbf{3 ~ \& ~ 5 ~}$ | Exhaust ports \{Port EA \& EB \} |
| $\mathbf{X}$ | External Pilot Supply (Used to supply pilot for low pressure or vacuum <br> applications) |
| $\mathbf{E X}$ | Pilot Exhaust (Never plug. Leave open or use a silencer) |



Each square represents a position or state that the valve will perform. The square that has the call outs will always show the valve at rest.

## At Rest Action

| 2 port NC | $\mathrm{P} \rightarrow$ Blocked | $\mathrm{A} \rightarrow$ Blocked |  |
| :---: | :--- | :--- | :--- |
| 2 port NO | $\mathrm{P} \rightarrow \mathrm{A}$ |  |  |
| 3 Port NC | $\mathrm{P} \rightarrow$ Blocked | $\mathrm{A} \rightarrow \mathrm{E}$ |  |
| 3 Port NO | $\mathrm{P} \rightarrow \mathrm{A}$ | $\mathrm{E} \rightarrow$ Blocked |  |
| 5 Port / 2 Position | $\mathrm{P} \rightarrow \mathrm{B}$ | $\mathrm{A} \rightarrow$ EA | EB $\rightarrow$ Blocked |
| 5 Port / 3 Position - Closed | $\mathrm{P}, \mathrm{B} \& \mathrm{~A} \rightarrow$ Blocked | $\mathrm{EA} \&$ EB $\rightarrow$ Blocked |  |
| 5 Port / 3 Position - Exhaust | $\mathrm{P} \rightarrow$ Blocked | $\mathrm{B} \rightarrow \mathrm{EB}$ | $\mathrm{A} \rightarrow$ EA |
| 5 Port / 3 Position - Open | $\mathrm{P} \rightarrow \mathrm{B} \& \mathrm{~A}$ | $\mathrm{EA} \& \mathrm{~EB} \rightarrow$ Blocked |  |

## Birctronticontrol Valves

## Valve Functions

A directional control valve determines the flow of air between its ports by opening, closing or changing its internal connections. The valves are described in terms of: the number of ports, the number of switching positions, its normal (not operated) position and the method of operation. The first two points are normally expressed in the terms $5 / 2,3 / 2,2 / 2$ etc. The first figure relates to the number of ports (excluding pilot ports) and the second to the number of positions.

The main functions and their ISO symbols are:

| Symbol | Principal Construction | Function | Application |
| :---: | :---: | :---: | :---: |
|  |  | 2/2 ON/OFF without exhaust. | Air motors and pneumatic tools |
|  |  | 3/2 Normally closed (NC), pressurizing or exhausting the output A | Single acting cylinders (push type), pneumatic signals |
|  |  | 3/2 Normally open (NO), pressurizing or exhausting the output A | Single acting cylinders (pull type), inverse pneumatic signals |
|  |  | 4/2 Switching between output $A$ and $B$, with common exhaust | Double acting cylinders |
|  |  | 5/2: Switching between output $A$ and $B$, with separate exhausts. | Double acting cylinders |
|  |  | 5/3, Open center: As $5 / 2$ but with outputs open to exhaust in midposition | Double acting cylinders, with the possibility to depressurize the cylinder |
|  |  | 5/3 Closed center: As 5/2 but with midposition fully shut off | Double acting cylinders, with stopping possibility |
|  |  | 5/3 Pressurized center: | Special applications, i.e. <br> Locking or Rodless Cylinder |

[^0]
## 

## Port Identification

The denominations or nomenclature used to identify the various ports was not uniform until the $5 / 2$ and $5 / 3$ valves were invented. Until the $5 / 2$ and $5 / 3$ were invented, there was more tradition than any respected standard.

Originally, the codes previously used for older hydraulic equipment were adopted. "P" for the supply port comes from "pump", the hydraulic source of fluid energy, and is understood to mean "pressure" in pneumatic systems.
The outlet of a $2 / 2$ (two ports, two positions) or $3 / 2$ valve has always been " $A$ ", with the second, antivalent output port labeled "B".
The exhaust port was originally labeled " R " from Return (to the oil tank). We can think of $R$ as return to atmosphere in pneumatic systems. The second exhaust port in $5 / 2$ valves was sometimes named $S$, or
the former "R1" and the latter "R2".
The pilot port initiating the power connection to port A has originally been coded "Z" (the two extreme letters in the alphabet belongs together) and the other " Y ".

After 20 years of bargaining about pneumatic and hydraulic symbols, one of the ISO work groups had the idea that ports should have numbers instead of letters, thus delaying the termination of the standard ISO 1219 by another 6 years. Supply should be " 1 ", the outputs " 2 " and " 4 ", the pilot port connecting " 1 " with " 2 " is then " 12 " etc. Table A shows the main sets of port identifications in use. Preferred are now the ISO 5599 numbers.

| Standard | Supply <br> Port | NC output | NO output | Exhaust of <br> NC | Exhaust of <br> NO | Pilot for NC | Pilot for NO |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Old JIS | P | A | B | R | S | Z | Y |
| ISO 1219 | P | A | B | R | S | Z | Y |
| JIS | P | A | B | R1 | R2 | Z | Y |
| JIS | 1 | 4 | 2 | 5 | 3 | 14 | 12 |
| NFPA | P | A | B | EA | EB | PA | PB |
| ISO 5599 | 1 | 4 | 2 | 5 | 3 | 14 | 12 |
| SMC | P(1) | A (4) | B (2) | EA (5) | EB (3) | PA (14) | PB (12) |

Table A Typical port identifications

## Monostable And Bi-stable

Spring returned valves are monostable (stable in one default or preferred condition). They have a defined preferred position to which they automatically return. A bi-stable valve has no preferred position and remains in either position until one of its two impulse signals are operated.

## Valve Types

The two principal methods of construction are Poppet and Slide with either elastic (rubber) or metal seals. Fig. B relates to the various combinations.


Fig. B The various types of valves and sealing methods

## Acronyms for Materials

| C3604 | Copper alloy per JIS H 3250 type C 3604 |
| :--- | :--- |
| CR | Neoprene |
| EPR | Ethylene-propylene rubber |
| FKM | SL |
| SPC | Cold roll steel |
| NBR | Buna N or Nitrile rubber |
| PBT | Polybutylene terephthalate |
| SUS | Stainless steel |
| POM | Polyacetal (Delrin) |
| PP | Poly-propylene |
| SUS304 | 304 grade stainless steel |
| Si | Silicone rubber |

Indication of International Standard Code for Production Lot No.


For example:
Production in Italy on November 1996 ...AYIT
Production in USA on May 2000 ... ESI


## Note)

1. Exception: Country code is not available for SMC- Japan, SMC-China and SMC Manufacturing (Singapore).
2. Exception: Country code is not available for SMC US, instead use [ I ] for Indianapolis factory.
3. If 2 or more production facilities will exist in future, add number of facilities after this code in order of registration.
4. In case of necessity of additional information, Job No. etc., add them after this code.

## Gyfider part Number Buiding information

- Style? $\qquad$
- Bore? $\qquad$
- Stroke? $\qquad$
- Single or Double Acting?
- Spring return or spring extend?

Mounting? $\qquad$ Inch or Metric?
Auto -Switch Capable? Y or N

- Number of Switches? $\qquad$
- Reed or Solid State? NPN or PNP?
- What Voltage? $\qquad$
- Standard or Long Leads?
- Prewired lead connector?

Options

- Oversize rod?
- Cushions? Air or Urethane?
- Non-rotating rod?
- Rod boot? Nylon or Neoprene?
- Low or High Temp application?
-Low Friction?
- Stainless Steel Rod?
- Adjustable Stroke? Extend or Retract?
- Dual Stroke? Single or Double Rod?
- Extended rod? Inch or Metric?
- Extended rod threads? Inch or Metric?
- Special Rod threads?


## Accessories

- Rod Eye
- Double Rod Clevis
- Flange (Head or Rear)
- Single Rod Clevis
- Foot Bracket
- Trunnion
$\qquad$
Temperature $\qquad$ Environment $\qquad$
Moments: X $\qquad$ Y $\qquad$ Z

[^1]Bore $\qquad$ Stroke $\qquad$ Inch or Metric Port Size $\qquad$

Thread Size $\qquad$ Mounting Style $\qquad$

Line Pressure $\qquad$ Load $\qquad$

Vertical or Horizontal Lift $\qquad$ Switches $\qquad$ Style $\qquad$

Dimensions:
A $\qquad$ B $\qquad$ C $\qquad$ D $\qquad$
E $\qquad$ F $\qquad$ G $\qquad$


Stroke $=$ G-C

- How Many Ports? $\qquad$
- How Many Positions? $\qquad$
- Flow? $\qquad$
- Rubber or Metal Seal? $\qquad$
- What is the application? $\qquad$

Cylinder bore? $\qquad$ Stroke? $\qquad$
$\qquad$

Speed? $\qquad$ Blow off? $\qquad$

- Single or Double Solenoid? $\qquad$
- Voltage? $\qquad$
- Style of Connector? $\qquad$

Plug-In, DIN or Grommet?

Serial or Discrete?

- Body Ported, Sub-plate or Manifold?
- Foot bracket, Mounting holes or DIN Rail?
- Port Size? $\qquad$ Threaded or One Touch Fitting
- How Many Stations? $\qquad$
- Operating Pressure? $\qquad$
- Temperature? $\qquad$
-Environment? $\qquad$
- Ejector - Single stage, 2-stage or 3-stage nozzle?
- Port size? $\qquad$
- Flow? $\qquad$
- Application:
- Horizontal or Vertical Lift?
- Load Material? $\qquad$
- Weight of Load? $\qquad$
- Number of Pads? $\qquad$
- Surface Material? $\qquad$
- Pad Diameter? $\qquad$
- Flat, Flat w/ Ribs, Deep or Bellows?
- Material? $\qquad$
- Connection - Vertical or Horizontal Vacuum entry?
- Buffer or Non - Buffer?
- Female Fitting, Barb or One-Touch?
- Vacuum Pressure? $\qquad$
- Vacuum Filter? $\qquad$
- Solenoid Valves for Supply and/or Blow off?
- Voltage? $\qquad$
- Type of connector, Grommet, L type, M type?
- Individual or Manifold?
- Vacuum Switch or Adsorption Conformation?
PNP or NPN?

REED SWITCHES: A thin metal contact is drawn closed by the magnetic field of the piston magnet. Since this is a mechanical switch it will wear out over time and is susceptible to vibration and shock. Their advantage is that they are inexpensive and can be used with AC voltages.
SOLID- STATE SWITCHES: The magnetic field generated by the piston magnet causes a current flow inside the switch. Since there are no moving parts, the switch life is much longer than a reed switch and they are less prone to vibration and shock. They are more expensive, can only be used with DC voltages and you need to know whether you need a sinking or sourcing switch.
Current Sinking (NPN) -The switch sensor "sinks" current from the load through the sensor to ground. The load is connected between the positive voltage supply and the output lead of the sensor.

## 3-Wire NPN Sensor Connection



Current Sourcing (PNP) - The switch sensor "Sources" current through load to ground. The load is connected between the output lead of the sensor and the negative "ground" lead of the supply.

## 3-Wire PNP Sensor Connection



Three wire DC sensors include one wire that provides voltage to the sensor, an output signal wire and a ground wire. Most electro-mechanical loads (relays, counters, solenoids etc.) can use either a sink or source type switch provided it is wired properly. The proper sensor type must be chosen when used with solid-state load and programmable controllers due to the fact that some of these loads must be grounded.
Wire Colors: SMC has changed the wire colors on all of our switch products. This was done to conform to European standards that are being adopted worldwide.
$\left.\begin{array}{ll}\text { Positive } & \text { Red } \\ \text { Negative } & \text { Black } \\ \text { Output } & \text { White }\end{array}\right\}$ (old colors) $\left.\begin{array}{ll}\text { Brown } \\ \text { Blue } \\ \text { Black }\end{array}\right\}$ (new colors)

## Pressure Switches and thetrestmplited Operation



Sourcing - PNP is often referred to as Sourcing, because the switch closes and provides the source voltage to the load

Sinking - NPN is often referred to as Sinking, because the switch closes and sinks the current to ground

Normally Open - Does not pass the signal until the set point is reached

Normally Closed - Passes current until the set point is reached

FS or Full Scale - The maximum setting minus the minimum setting.

Ex. ITV1050 0.9MPa - 0.005MPa $=0.895 \mathrm{MPa}$ Full Scale ( $130.5 \mathrm{psi}-0.725 \mathrm{psi}=129.775 \mathrm{psi})$
Linearity - The nearness with which the plot of a signal, or variable, plotted against a prescribed linear scale approximates a straight line. Output error to reference value

Repeatability - The ability of the instrument to provide the same output every time for the same input. Usually given as a \% of the FS value

Sensitivity - Often described as the minimum change of input to which the system is capable of responding. Usually expressed in \% of Full Scale

Hysteresis - The difference in output when the measured value is first approached with increasing and then decreasing values. Expressed in \% of Full Scale

Impedance - Resistance of a load that hinders the flow.
Current Consumption - The amount of current needed for normal operation, does not include load current.

Watts (W) and Volt Amps (VA) - Both of these units are used to express electrical power.

Watts is for DC voltage and Volt Amps is for AC voltage.
If you have any questions on basic electronics there is an entry in the Product Application Database that explains basic electronics.


Linearity

This graph shows the repeatability of an analog output, pressure display and a switch (ON-OFF) output's moving point. The pressure is increased or decreased under normal temperature ( $77^{\circ} \mathrm{F}\left(25^{\circ} \mathrm{C}\right)$ ).


Repeatability


Hysteresis

An enclosure is a surrounding case constructed to provide a degree of protection to personnel against accidental contact with the enclosed equipment and to provide a degree of protection to the enclosed equipment against specified environmental conditions. These are the more common classifications as they pertain to pneumatic components such as valves.

NEMA 1 Intended for Indoor use primarily to provide a degree of protection against contact with enclosed equipment.

NEMA 2 Intended for indoor use primarily to provide a degree of protection against limited amounts of falling water and dirt.

NEMA 3 Intended for outdoor use to provide a degree of protection against windblown dust, rain, sleet and external ice formation.

NEMA 3R Intended for outdoor use to provide a degree of protection against falling rain, sleet and external ice formation.

NEMA 3S Intended for outdoor use to provide a degree of protection against windblown dust, rain, sleet and provide for operation of external mechanisms when ice laden.

NEMA 4 Intended for indoor and outdoor use primarily to provide a degree of protection against windblown dust and rain, splashing water and hose directed water.

NEMA 4X Intended for indoor and outdoor use primarily to provide a degree of protection against corrosion, windblown dust and rain, splashing water and hose directed water.

NEMA 6 Intended for indoor or outdoor use primarily to provide a degree of protection against entry of water during occasional submersion to a limited depth.

## 

| $1^{\text {st }}$ Numeral: <br> Degree of protection with respect to persons and solid objects |  | ${ }^{2 \text { nd }}$ Numeral: |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  | $\begin{aligned} & \frac{\pi}{0} \\ & \frac{0}{\omega} \\ & \frac{0}{0} \\ & 3 \end{aligned}$ |  |  |  |
| Not protected | 0 | IP00 | IP01 | IP02 |  |  |  |  |  |  |
| Solid objects > $\varnothing 50 \mathrm{~mm}$ | 1 | IP10 | IP11 | IP12 | IP13 |  |  |  |  |  |
| Solid objects > $\quad 12 \mathrm{~mm}$ | 2 | IP20 | IP21 | IP22 | IP23 |  |  |  |  |  |
| Solid objects $>\varnothing$ ¢ 2.5 mm | 3 | IP30 | IP31 | IP32 | IP33 | IP34 |  |  |  |  |
| Solid objects > $\varnothing 1.0 \mathrm{~mm}$ | 4 | IP40 | IP41 | IP42 | IP43 | IP44 | $\begin{aligned} & \text { IP } \\ & 45 \end{aligned}$ | IP 46 |  |  |
| Dust protected | 5 |  |  |  |  | IP54 | $\begin{aligned} & \text { IP } \\ & 55 \end{aligned}$ | $\begin{aligned} & \text { IP } \\ & 56 \end{aligned}$ |  |  |
| Dust tight | 6 |  |  |  |  |  | $\begin{aligned} & \hline \text { IP } \\ & 65 \\ & \hline \end{aligned}$ | $\begin{aligned} & \hline \text { IP } \\ & 66 \\ & \hline \end{aligned}$ | $\begin{aligned} & \hline \mathrm{IP} \\ & 67 \\ & \hline \end{aligned}$ | $\begin{aligned} & \hline \mathrm{IP} \\ & 68 \\ & \hline \end{aligned}$ |

Note: find IP rating and follow across and up to find degree of combined protection. IP65 and NEMA 4 are roughly equivalent

## Threaded Type KQ2 H 05-34 A S <br> One-Touch Fittings <br> Series KQ2

| Model |  |
| :---: | :---: |
| Symbol | Model |
| H | Male connector |
| S | Hexagon socket head male connector |
| F | Female union |
| L | Male elbow |
| K | $45^{\circ}$ male elbow |
| V | Universal male elbow |
| VS | Hexagon socket head universal male elbow |
| VF | Universal female elbow |
| LF | Female elbow |
| VD | Double universal male elbow |
| VT | Triple universal male elbow |
| Z | Branch universal male elbow |
| ZD | Double branch universal male elbow |
| ZT | Triple branch universal male elbow |
| W | Extended male elbow |
| T | Male branch tee |
|  | Union Tee |
|  | Different diameter Tee |
| Y | Male run tee |
| U | Branch " Y " |
|  | Union " Y " |
|  | Different diameter Union "Y" |
| X | Different diameter plug-in "Y" |
| E | Bulkhead union |
|  | Bulkhead connector |
| LE | Bulkhead union elbow |
| N | Adaptor |



Thread material/Surface treatment

| Symbol | Thread material/Surface treatment |
| :---: | :---: |
| $\mathbf{A}$ | Brass (compatible with KQE) |
| $\mathbf{N}$ |  |
| Brass + Electroless nickel plated <br> Compatible to KQE-X2 |  |
| Bulkhead <br> union | $\square \mathbf{J}$ |

* $\square / \mathrm{A}, \mathrm{N}$
- Port size/Applicable tubing O.D.

| Symbol |  | Size |
| :---: | :---: | :---: |
| Thread connection | 32 | 10-32UNF |
|  | 33 | NPT1/16 |
|  | 34 | NPT1/8 |
|  | 35 | NPT1/4 |
|  | 36 | NPT3/8 |
|  | 37 | NPT1/2 |
| Tubing connection | 00* | Same diameter tubing |

- Applicable tubing O.D.

| Symbol | Size |
| :---: | :---: |
| 01 | $\varnothing 1 / 8^{\prime \prime}$ |
| 03 | $\varnothing 5 / 32^{\prime \prime}$ |
| 05 | $\varnothing 3 / 16^{\prime \prime}$ |
| 07 | $\varnothing 1 / 4^{\prime \prime}$ |
| 09 | $\varnothing 5 / 16^{\prime \prime}$ |
| 11 | $\varnothing 3 / 8^{\prime \prime}$ |
| 13 | $\varnothing 1 / 2^{\prime \prime}$ |

## Spare Parts

Use the part number below to order the gasket for sealing
10-32UNF thread
Gasket for 10-32UNF: M-10/32G

## Tube - Tube Type KQ2 H 05-00 A




| 1. Filter | Port Size | Part No. W/ Manual Drain | Part No. W/Auto Drain |  |
| :---: | :---: | :---: | :---: | :---: |
|  | 1/8" NPT | AF20-N01-CZ-A | AF20-N01C-CZ-A |  |
|  | 1/4"NPT | AF20-N02-CZ-A | AF2O-N02C-CZ-A |  |
|  | 3/8" NPT | AF30-N03-Z-A | AF30-N03D-Z-A |  |
|  | 1/2" NPT | AF40-N04-Z-A | AF40-N04D-Z-A |  |
|  | 3/4" NPT | AF50-N06-Z | AF50-N06D-Z |  |
|  | 1"NPT | AF60-N10-Z | AF60-N10D-Z |  |
| 2. Regulator | Port Size | Part Number W/O gauge | Part Number W/gauge |  |
|  | 1/8" NPT | AR20-N01H-Z-A | AR20-N01GH-Z-A |  |
|  | 1/4" NPT | AR20-N02H-Z-A | AR2O-N02GH-Z-A |  |
|  | 3/8" NPT | AR30-N03H-Z-A | AR30-N03GH-Z-A |  |
|  | 1/2" NPT | AR40-N04H-Z-A | AR40-N04GH-Z-A |  |
|  | 3/4" NPT | AR50-N06H-Z | AR50-N06GH-Z |  |
|  | 1"NPT | AR60-N10H-Z | AR60-N10GH-Z |  |
| 3. Lubricator |  | Port Size | Part Number |  |
|  |  | 1/8" NPT | AL20-N01-3CZ-A |  |
|  |  | 1/4" NPT | AL2O-NO2-3CZ-A |  |
|  |  | 3/8" NPT | AL30-N03-3Z-A |  |
|  |  | 1/2" NPT | AL40-N04-3Z-A |  |
|  |  | 3/4" NPT | AL50-N06-3Z |  |
|  |  | 1"NPT | AL60-N10-3Z |  |
| 4. Brackets |  | Air Prep Unit Port Size | Spacer | Spacer-T |
|  |  | 1/8" NPT (AC20 Series) | Y200-A | Y200T-A |
|  |  | 1/4" NPT (AC20 Series) | Y200-A | Y200T-A |
|  |  | 3/8" NPT (AC30 Series) | Y300-A | Y300T-A |
|  |  | 1/2" NPT (AC40 Series) | Y400-A | Y400T-A |
|  |  | /4" NPT (AC50 Series) | Y500-A | Y500T-A |
|  |  | "NPT (AC60 Series) | Y600 | Y600T |


[^0]:    Valve Symbols, Principles, description and main applications

[^1]:    Note: Use cylinder dimensional sketch on page 19, if necessary.

